STANDARD SP CLUTCH SP114P

QUALITY IS STANDARD

- TAPERED ROLLER MAIN BEARINGS
- AVAILABLE IN SIZES 11.5" THRU 21.0"
- OPTIONAL BEARING THORW OUT
- OPTIONAL SINTERED IRON PLATES
- BUILT IN HEX NUT
- · CREATES SUITABLE APPLICATION TORQUE CAPACITY
- MORE SUITABLE FOR SIDE LOAD APPLICATIONS
- ALLOWS FOR MORE FREQUENT ENGAGEMENTS
- · CREATES 25% HIGHER TORQUE CAPACITY
- · EASES ADJUSTMENT VERIFICATION





SPECIFICATIONS - SP114P

	OI LOII IOAIII	0. 11	••							
	Model	SAE HSG.	Max. Input Torque Nm (lb-ft)							
	Number				Solid Plates		Split Plates		Weight	
			Organic	Sintered	Cast Drive Ring	Nodular Drive Ring	Cast Drive Ring	Nodular Drive Ring	kg (lbs)	
	SP114P1	1	1099 (810)	1370 (1010)	2400	3000	1950	2750	118 (260)	
	SP114P0	0								

LOAD CLASSIFICATIONS BASED UPON AGMA LOAD CHARACTERISTICS

PRIME MOVER	DURATION	DRIVEN MACHINE LOAD CLASSIFICATIONS							
PRIME MUVER	OF SERVICE	UNIFORM	MODERATE SHOCK	HEAVY SHOCK					
Electric motor	Up to 3 hours per day	1.00	1.25	1.50					
	3-10 hours per day	1.00	1.25	1.75					
	Over 10 hours per day	1.25	1.50	2.00					
Multi-cylinder internal combustion engine	Up to 3 hours per day	1.00	1.25	1.75					
	3-10 hours per day	1.25	1.50	2.00					
	Over 10 hours per day	1.50	1.75	2.25					
Multi-cylinder internal combustion engine with high torque rise	Up to 3 hours per day	1.50	1.75	2.25					
	3-10 hours per day	1.75	2.00	2.50					
	Over 10 hours per day	2.00	2.25	2.75					
Single cylinder internal combustion engine	Up to 3 hours per day	1.25	1.50	2.00					
	3-10 hours per day	1.50	1.75	2.25					
	Over 10 hours per day	1.75	2.00	2.50					

All clutch engagements to be with prime mover below 1000 RPM. High inertia loads may require use of larger clutch. Contact Twin Disc application engineering department for assistance.

TO CALCULATE APPLICATION TORQUE:

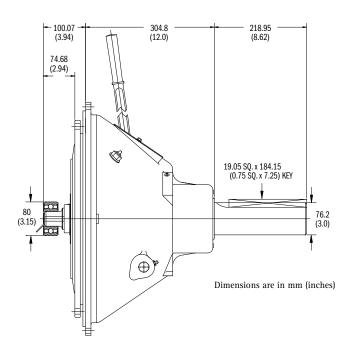
 $\frac{5252 \text{ x HP}}{\text{Engine RPM}} = \text{Torque}$

Torque x Load Factor = Application Torque

Use load factor from chart at left



SP114P



SP114P - ALLOWABLE SIDE LOAD, KG (LBS)

01 1141	of 1141 ALLOWADEL SIDE ESAD, RG (LDS)								3
PTO	RPM	X DISTANCE, mm (in) – see sketch							
MODEL		25.4 (1.0)	50.8 (2.0)	76.2 (3.0)	101.6 (4.0)	127.0 (5.0)	152.4 (6.0)	177.8 (7.0)	203.2 (8.0)
	1000	0 1538 (3390)	1179 (2600)	962 (2120)	807 (1780)	696 (1535)	612 (1350)	549 (1210)	494 (1090)
SP114P0	1500								
SP114P1	2000								
	2200								

The following general formula should be used for determining the actual applied load: $L = \frac{126,000 \text{ x HP}}{\text{N x D}} \text{ x F x LF}$

WHERE L = Actual Applied Load (lbs)

N = Shaft Speed (RPM)
D = Pitch Diameter (in) of Sheave, etc.

F = Load Factor

F = Load Factor

1.0 for Chain or Gear Drive, 1.5 for Timing Belts, 2.5 for All V Belts, 3.5 for Flat Belts

LF = 2.1 for Reciprocating Compressors and other Severe Shock Drives and 1.8 for Large Inertia

Type Drives (i.e. crushers, chippers, planers, etc.)

Compound Drives and Power Engaged Power Take-Off applications must have written factory review.

Twin Disc, Incorporated reminds users of these products that their safe operation depends on use in compliance with engineering information provided in our catalog. Users are also reminded that safe operation depends on proper installation, operation and routine maintenance and inspection under prevailing conditions. It is the responsibility of users (and not Twin Disc, Incorporated) to provide and install guards or safety devices which may be required by recognized safety standards or by the Occupational Safety and Health Act of 1970 and its subsequent provisions.

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